## These are sample MCQs to indicate pattern, may or may not appear in examination

# University of Mumbai Online Examination 2020 

Program: BE Automobile Engineering

Curriculum Scheme: Revised 2016
Examination: Fourth Year (Semester VII)
Course Code: AEC701 and Course Name: Automotive Design
Time: 1 hour
Note to the students: All the Questions are compulsory and carry equal marks.

| Q1. | The width of space between the two adjacent teeth measured along the pitch <br> circle is known as |
| :--- | :--- |
| Option A: | Working depth |
| Option B: | Tooth space |
| Option C: | Tooth thickness |
| Option D: | Total depth |
|  |  |
| Q2. | The overall heat transfer coefficient under normal working condition with <br> natural air circulation is |
| Option A: | 12 to $18 \mathrm{~W} / \mathrm{m}^{2}{ }^{\circ} \mathrm{C}$ |
| Option B: | 22 to $26 \mathrm{~W} / \mathrm{m}^{2}{ }^{\circ} \mathrm{C}$ |
| Option C: | 24 to $28 \mathrm{~W} / \mathrm{m}^{2} \mathrm{C}$ |
| Option $\mathrm{D}:$ | 20 to28 $\mathrm{W} / \mathrm{m}^{2}{ }^{\circ} \mathrm{C}$ |
|  |  |
| Q3. | In finding the tooth size and strength, it is safe to assume that the teeth of worm <br> gear are always <br> than the threads of the worm |
| Option A: | stronger |
| Option B: | equal |
| Option C: | weaker |
| Option D: | doubled |
|  |  |
| Q4. | In helical gears, the distance between similar faces of adjacent teeth along a <br> helix on the pitch cylinders normal to the teeth, is called |
| Option A: | normal pitch |
| Option B: | axial pitch |
| Option C: | diametral pitch |
| Option D: | module |
|  |  |
| Q5. | The form factor of a spur gear tooth depends upon |
| Option A: | circular pitch only |
| Option B: | pressure angle only |
| Option C: | number of teeth and circular pitch |


| Option D: | number of teeth and the system of teeth |
| :---: | :---: |
| Q6. | The connecting rods are made of I sections because |
| Option A: | It is easy to draw the assembly drawing |
| Option B: | It reduces the friction and inertia forces |
| Option C: | It reduces the friction and wear and tear |
| Option D: | It reduces the weight and inertia forces |
| Q7. | The values of thermal conductivity factor ( $k$ ) for grey cast iron is $\mathrm{k}=46.6 \mathrm{~W} / \mathrm{m} /{ }^{\circ} \mathrm{C}$, whereas for aluminium alloy it is $\qquad$ |
| Option A: | $175 \mathrm{~W} / \mathrm{m} /{ }^{\circ} \mathrm{C}$ |
| Option B: | $165 \mathrm{~W} / \mathrm{m} /{ }^{\circ} \mathrm{C}$ |
| Option C: | $155 \mathrm{~W} / \mathrm{m} /{ }^{\circ} \mathrm{C}$ |
| Option D: | $185 \mathrm{~W} / \mathrm{m} /{ }^{\circ} \mathrm{C}$ |
| Q8. | Cylinders and cylinder liners are usually made of $\qquad$ with homogeneous and close-grained structure. |
| Option A: | brass |
| Option B: | copper |
| Option C: | grey cast iron |
| Option D: | Forged steel |
| Q9. | The clearance between the cylinder liner and piston is provided to take care of $\qquad$ and distortion under load. |
| Option A: | thermal expansion |
| Option B: | linear expansion |
| Option C: | lateral expansion |
| Option D: | corrosion |
| Q10. | The length of the crank pin is determined by .................. consideration |
| Option A: | compression |
| Option B: | tension |
| Option C: | shearing |
| Option D: | bearing |
| Q11. | The material used for lining of friction surfaces of a clutch should have $\qquad$ coefficient of friction. |
| Option A: | low |
| Option B: | high |
| Option C: | very low |
| Option D: | very high |
| Q12. | In a centrifugal clutch, the force with which the shoe presses against the driven member is the $\qquad$ of the centrifugal force and the spring force. |
| Option A: | Difference |
| Option B: | Sum |


| Option C: | ratio |
| :---: | :---: |
| Option D: | product |
| Q13. | Total frictional torque acting on the friction surface in case of design of clutch is given by |
| Option A: | n. $\mu . \mathrm{w} . \mathrm{R}^{2}$ |
| Option B: | n. $\mu . \mathrm{w} . \mathrm{R}$ |
| Option C: | n. $\mu . \mathrm{w}$. |
| Option D: | U.w.R |
| Q14. | A plate clutch having a single driving plate with contact surfaces on each side is required to transmit 110 kW at $1250 \mathrm{r} . \mathrm{p} . \mathrm{m}$. The outer diameter of the contact surfaces is to be 300 mm . The coefficient of friction is 0.4 . find the required torque. |
| Option A: | $820 \mathrm{~N}-\mathrm{m}$ |
| Option B: | $840 \mathrm{~N}-\mathrm{m}$ |
| Option C: | $860 \mathrm{~N}-\mathrm{m}$ |
| Option D: | $880 \mathrm{~N}-\mathrm{m}$ |
| Q15. | The heat generation in brake depends upon |
| Option A: | p.v |
| Option B: | $\mathrm{p} / \mathrm{v}$ |
| Option C: | pv/2 |
| Option D: | $1 / 2 p v^{2}$ |
| Q16. | A band brake consists of the lever attached to one end of the band. The other end of the band is fixed to the ground. The wheel has a radius of 200 mm and wrap angle of the band is $270^{\circ}$. The braking force applied to the lever is limited to 100 N , and the coefficient of friction between the band and the wheel is 0.5 . No other information is given. The maximum wheel torque that can be completed is in Nm . It is assumed that the distance form pivot to fixed is 1 m and fixed to force is 1 m |
| Option A: | 200 |
| Option B: | 382 |
| Option C: | 604 |
| Option D: | 844 |
| Q17. | When the intensity of the pressure between the block and brake drum is uniform, the angle of contact between the block and brake drum is less than |
| Option A: | 45 |
| Option B: | 90 |
| Option C: | 60 |
| Option D: | 30 |
| Q18. | Braking torque for double block shoe brake is |
| Option A: | (Ft1+Ft2)r |
| Option B: | (Ft1+Ft2) |


| Option C: | Ft1 |
| :--- | :--- |
| Option D: | Ft2 |
|  |  |
| Q19. | The type of cam does not require any external force to have contact between <br> cam and follower is |
| Option A: | cylindrical cam |
| Option B: | conjugate cam |
| Option C: | disk cam |
| Option D: | end cam |
|  |  |
| Q20. | The push rod is designed by considering it as |
| Option A: | beam |
| Option B: | shaft |
| Option C: | column |
| Option D: | axle |
|  |  |
| Q21. | The push rod is located between the |
| Option A: | valve spring and valve |
| Option B: | rocker arm and valve |
| Option C: | rocker arm and cam shaft |
| Option D: | rocker arm and tappet |
|  |  |
| Q22. | Cams with translating followers are designed with pressure angle of |
| Option A: | 25 degree |
| Option B: | 20 degree |
| Option C: | 30 degree |
| Option D: | 40 degree |
|  |  |
| Q23. | The velocity ratio of two pulleys connected by an open belt or crossed belt is |
| Option A: | directly proportional to their diameters |
| Option B: | inversely proportional to their diameters |
| Option C: | directly proportional to the square of their diameters |
| Option D: | inversely proportional to the square of their diameters |
|  |  |
| Q24. | Calculate the angle of wrap if diameter of the two pulleys are 550 mm and <br> 300 mm . Also the centre distance is 2800mm. <br> Option A: |
| Option B: | $174.8^{\circ}$ |
| Option C: | $167.8^{\circ}$ |
| Option D: | $200^{\circ}{ }^{\circ}$ |
| Option C: | They are not endless and hence not smooth motion |
| Q25. | They are endless and gives noise |
| Option B: | Yes |
| No they are very noisy |  |

